

Preliminary MRA Structural Analysis

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Mar 26, 2024

ORNL is managed by UT-Battelle, LLC for the US Department of Energy



Structural analysis for MRA Reflector Vessels and Backbone

- Separate CFD thermal analysis* was done for the reflector vessels and Stainless Steel Backbone
- Three Abaqus Structural models were made for the upper reflector vessel, the lower reflector vessel and the backbone
- Temperature profiles from the CFD analysis were imported and mapped onto the Abaqus models

*Temperature Profiles from Min-Tsung Kao – Reflector Vessel Thermal hydraulic Analysis



Reflector Vessel Analysis

- This structural analysis of the upper and lower reflector vessels included the temperature distributions from thermal hydraulic analysis
- The analysis was done for 5 different load cases which could occur during operation plus an evaluation of the thermal stress alone
- The analysis identified the minimum distance between the allowable membrane plus bending stress and the closest weld for both vessels
- Stresses were compared to the ASME BPVC code section 8 Division 2 allowable limits
- Peak deflections towards the target were found and compared to the limit of 1 mm



Material Properties for Structural analysis

Aluminum 6061T6 properties

Modulus of Elasticity (GPa)	68.9
Poisson's Ratio	0.33
Coefficient of Thermal Expansion (μ/°C)	23.6
(100° F) Sm (MPa)	84.8 (12.3 ksi)
Sm Weld (MPa)	55 (8ksi)

ASME BPVC Section8 Division 2 Allowable equivalent stress values

Code case 2478-1

	<u>Cree</u>	Sm	Sm Membrane+
	Sm Membrane	Membrane+ Bending (1.5x)	Bending + Secondary (3x)
Non-Weld Regions			
(MPa)	84.8 MPa	127 MPa	254 MPa
Weld regions	55 MPa	83 MPa	165 MPa

Temperature not exceeding 93° C

Beryllium Properties

Modulus of Elasticity (GPa)	303
Poisson's Ratio	0.24
Coefficient of Thermal Expansion (μ/°C)	11.5
Tensile Yield Strength (MPa)	345 VHP-517 HIP



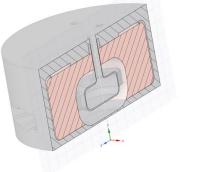
Weld Heat Affected Zones

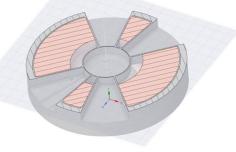
 ASME BPVC lists heat affected zone (HAZ) width of welds at 50 mm, 25 mm either side of the weld. The GTAW (arc) welds have a HAZ width up to 10 mm, correlating to 5 mm to either side of the centerline of welds. Stresses at or exceeding the allowable stress values for weld regions, detailed previously, were evaluated at half of these respective width values from the centerline of the weld.



MRA Upper Reflector Vessel

SpaceClaim model for Aluminum and Beryllium

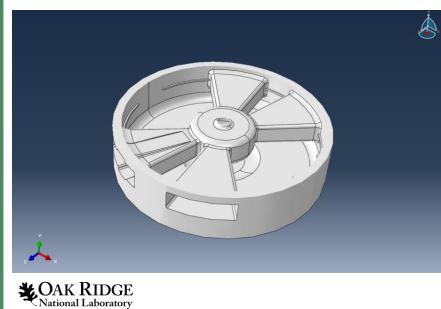


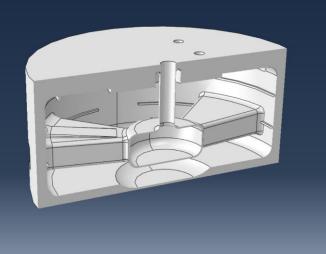


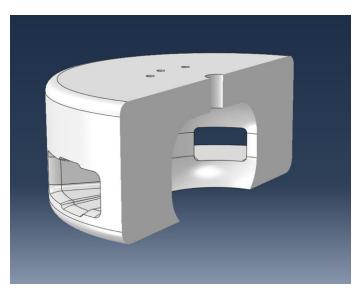
View with a cut above moderator vessel

Planar cut showing cylindrical moderator vacuum shell

Beryllium part with x plane cut







Upper Vessel Model mesh

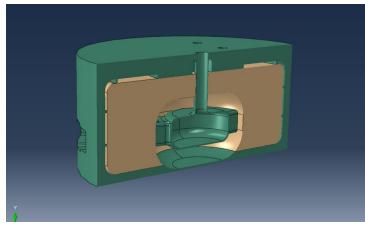
Aluminum

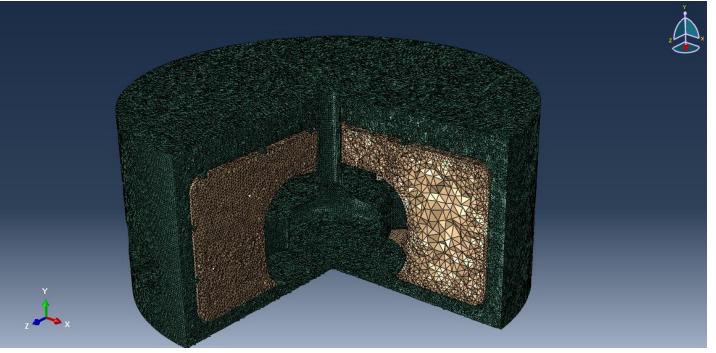
- C3D10 elements
- 1.9 mm nominal edge
- 2.5 mm on top and bottom outside edges
- .05 Deviation
- 1.05 Growth
- 6,565,043 elements

• Beryllium

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- C3D10 elements
- 3 mm nominal edge
- .1 deviation
- 1.1 Growth
- 1,171,786 elements



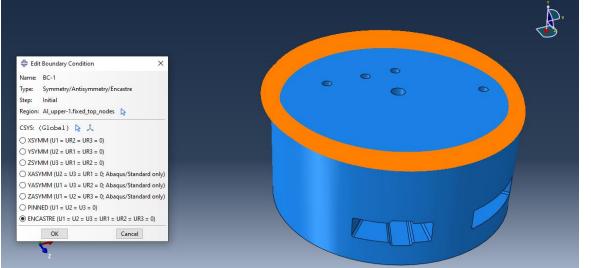


CAK RIDGE Aluminum Reflector Vessel pressure loads

Water pressure y-cut	Vacuum shell	Beryllium pressure	Core Vessel Pressure
(5 bar or 3 bar)	(0 or 2 bar)	(5 bar or 3 bar)	0 or 1.5 bar)

Boundary condition and constraints

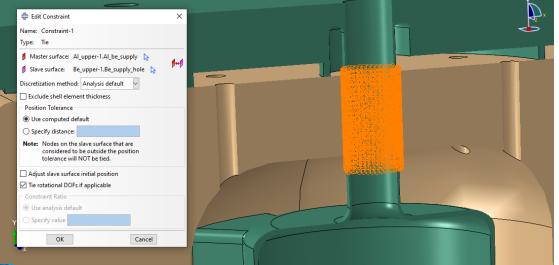
Fixed boundary condition to mimic bolting to backbone



Top Surface nodes in ring from 180.5 mm radius to outside at 190.5 mm fixed



Tie constraint



Artificial tie constraint used to keep beryllium centered within vessel



Load Cases

Case #	Load Case	Reflector P (bar)	Vac Vessel P (bar)	Core Vessel P (bar)	Thermal
1	Reflector MAWP	5	0	0	off
2	Vac Vessel MAWP	3	2	0	off
3	Core Vessel MAWP	3	0	1.5	off
4	Thermal profile	0	0	0	on
5	normal ops	3	0	0	on
6	Reflector MAWP + thermal	5	0	0	on

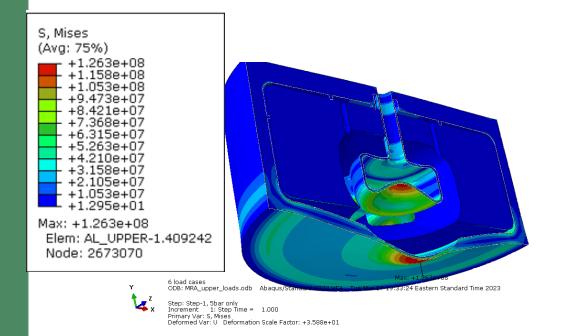
Results will be shown for Load Cases 1 and 6 which are limiting

MAWP = Maximum Allowable Working Pressure

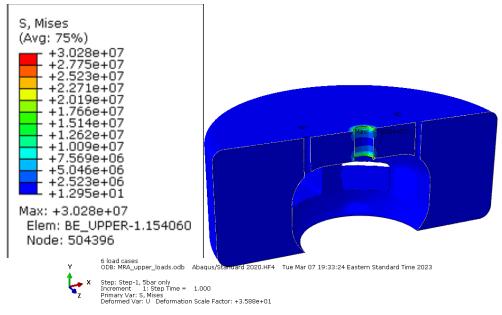


Upper Case 1 – 5 bar only

S peak 126.3 MPa



Beryllium S Mises peak 30.3 MPa by edge of tie to aluminum tube

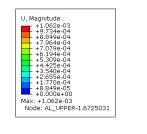


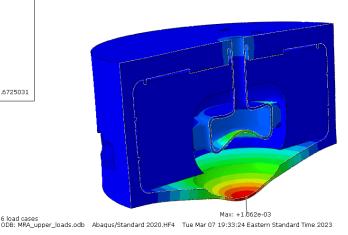


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Upper Case 1 – 5 bar only - Displacements

Displacement peak 1mm

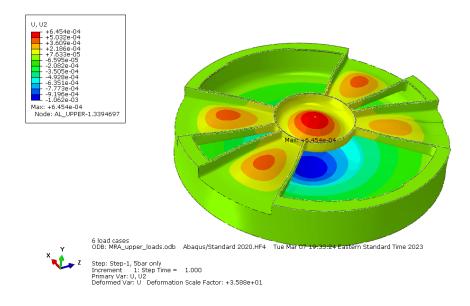




6 load cases

Step: Step-1, 5bar only Increment 1: Step Time = 1.000 Primary Var: U, Magnitude Deformed Var: U Deformation Scale Factor: +3.588e+01

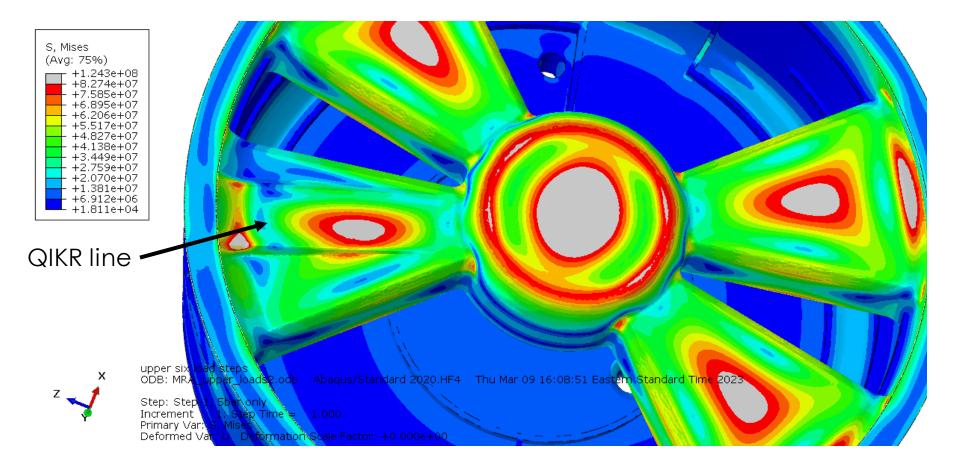
Vertical Displacement with cut through beam line channels



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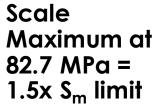
Upper case 1 5 bar 82.74 MPa scale max

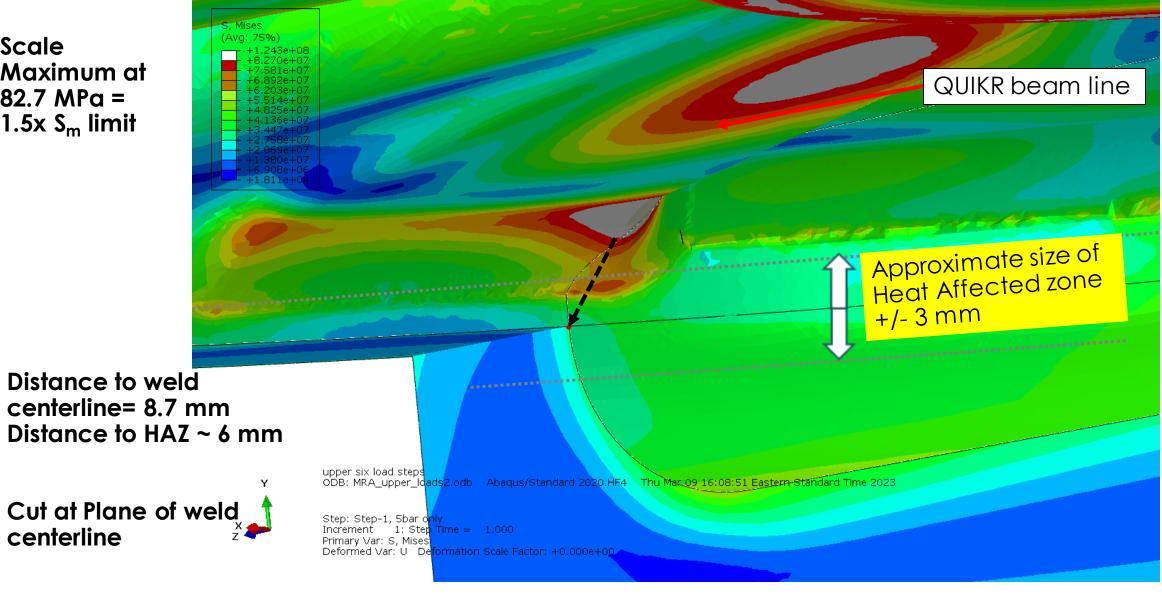


Scale Maximum at 82.7 MPa = 1.5x S_m limit for weld region



Upper Case1 Minimum distance from stress limit to weld

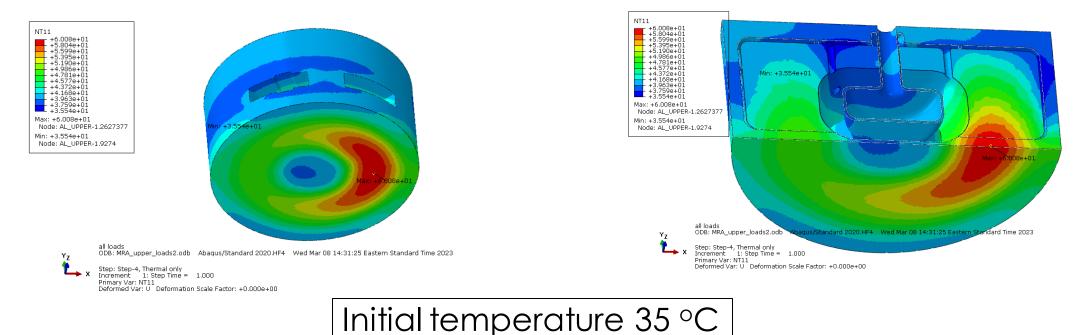




centerline

Upper Imposed Temperature fields

Outer surface peak 60 C



Vertical cut z=0

Temperature Profiles from Min-Tsung Kao – Reflector Vessel Thermal hydraulic Analysis

Min-Tsung Kao STS MRA 2023 01 18 Upper MRA.



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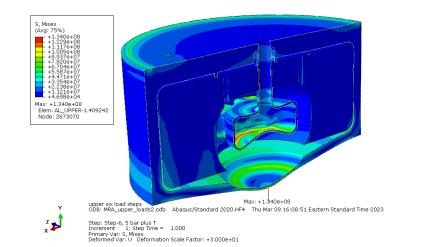
Upper Case 6 5 bar plus thermal

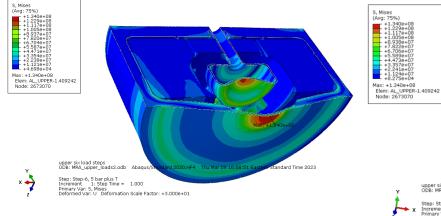
S Mises peak 134 MPa

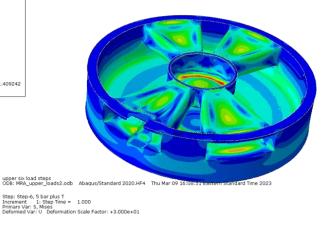
Bottom view

Stress with view cut above neutron beam channels

. Mises



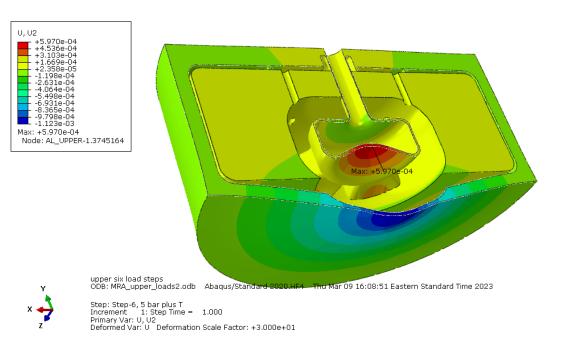




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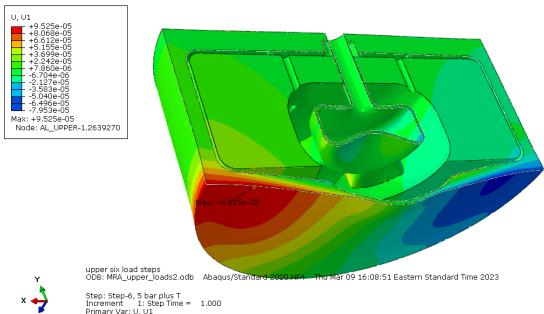
Upper Case 6 5 bar plus thermal Displacement

Vertical displacement max -1.1 mm Horizontal displacement U1 (x)



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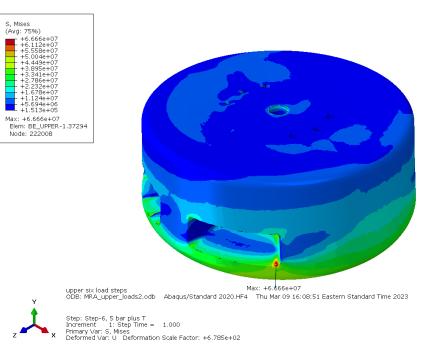
17



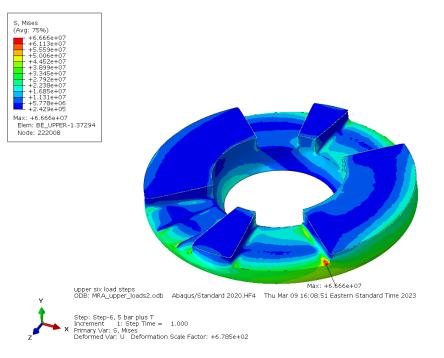
Deformed Var: U Deformation Scale Factor: +3.000e+01

Upper Case 6 5 bar plus thermal - Beryllium

Beryllium Von Mises peak 66.7 MPa



Beryllium y cut to show beam lines

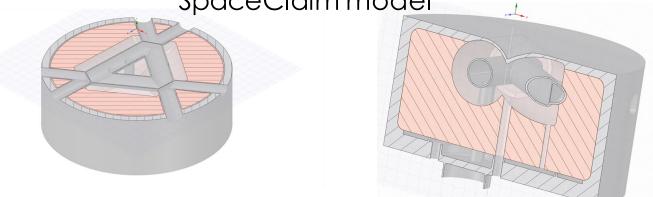




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Lower MRA Reflector

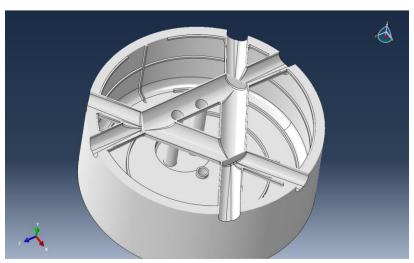
SpaceClaim model

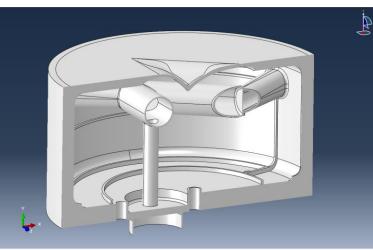


Horizontal cut through hydrogen vacuum vessel tubes

Vertical cut z=0 showing one 1mm vacuum window

Beryllium Part





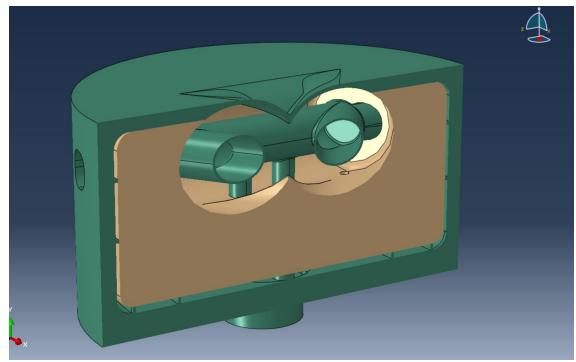




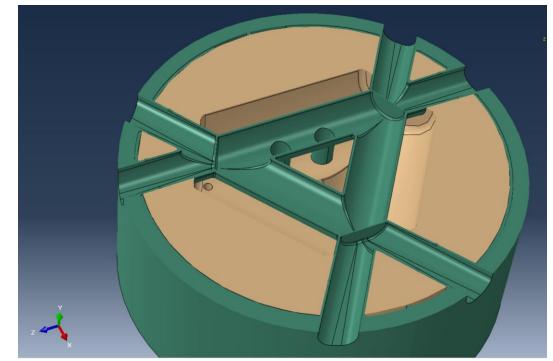
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Assembly Views

Vertical cut x=0



Horizontal cut y=-.0992m





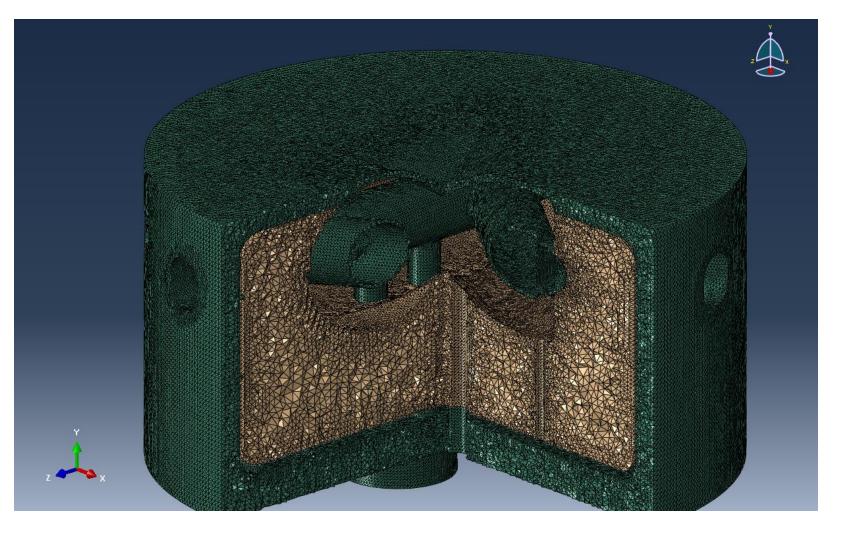
Lower MRA Mesh

• Aluminum

- C3D10 elements
- 2.0 mm nominal edge
- 3.5 mm on top and bottom outside edges
- .1 Deviation
- 1.05 Growth
- 3,312,375 elements

• Beryllium

- C3D10 elements
- 2 mm nominal edge
- 3 mm on top and bottom outside edges
- .1 deviation
- 1.05 Growth
- 2,364,100 elements



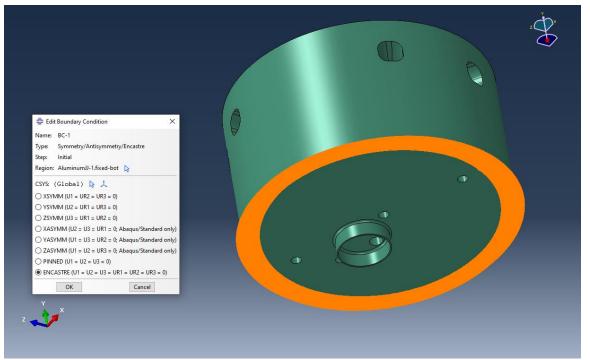


CAK RIDGE Pressure loads for Lower MRA

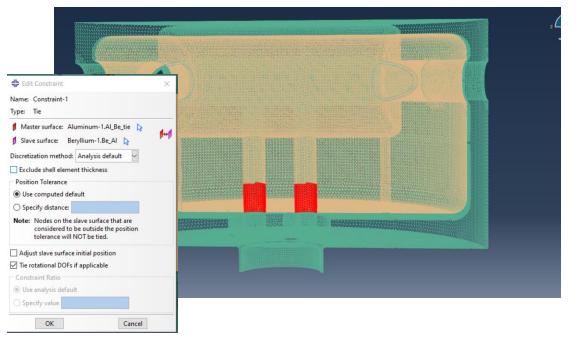
Water Pressure surface	Vacuum vessel surface	Beryllium pressure surface	Aluminum outer surface
5 bar and 3 bar cases	0 or 2 bar cases	5 bar and 3 bar cases	0 or 1.5 bar
<image/>			

Boundary Condition and Constraints

Fixed Boundary condition on bottom surface nodes



Tie constraint between Be and Al on tubes



Artificial tie constraint used to keep beryllium centered within vessel



Lower MRA Load Cases

Case #	Load Case	Reflector P (bar)	Vac Vessel P (bar)	Core Vessel P (bar)	Thermal
1	Reflector MAWP	5	0	0	off
2	Vac MAWP	3	2	0	off
3	CV MAWP	3	0	1.5	off
4	Thermal	0	0	0	on
5	normal ops	3	0	0	on
6	Reflector +thermal	5	0	0	on

Only Cases 1 and 6 which are limiting to be presented



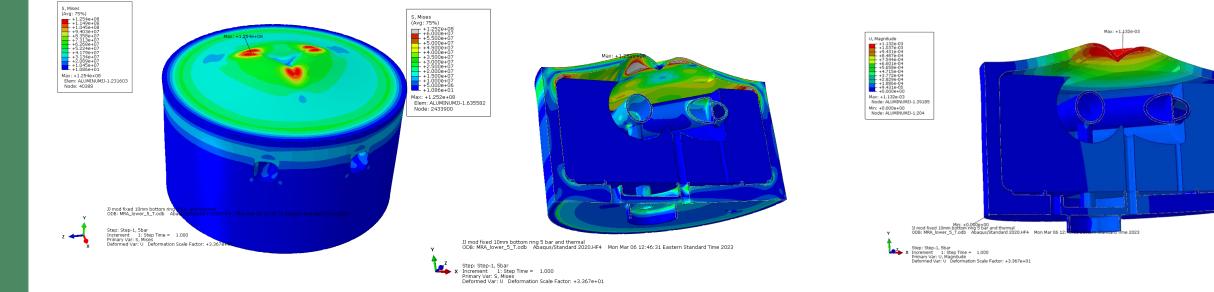
Lower Case 1 - 5 bar

S Mises peak 125.4 MPa

Z=0 cut with 60 MPa scale maximum

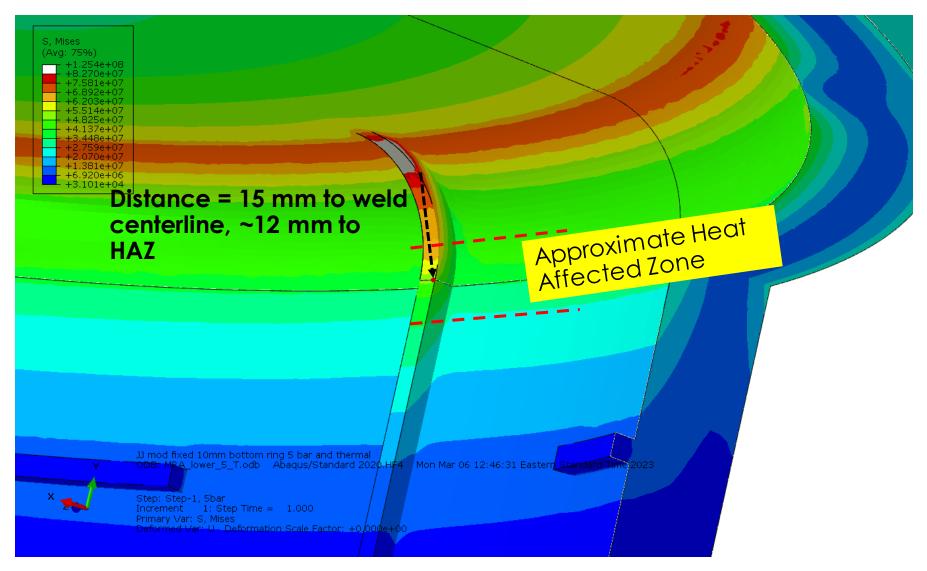
Displacement peak 1.1

mm





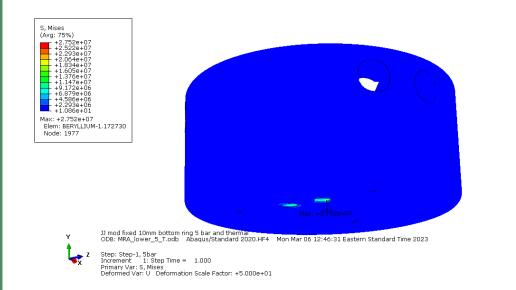
Lower Case 1 Minimum distance from weld to stress limit



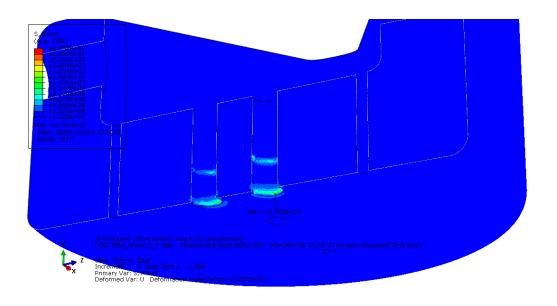


Lower Case 1 - 5 bar Beryllium

Beryllium S Mises peak 27.5 MPa



Cut showing peak around tie area





Lower Temperature Profile

Outer Al surface peak 54.1C

NT11 NT11 4.796e+0 +4.642e+0 +4.364e+0 20e+0 3.566e+0 Max: +5.410e+01 +3.766e+01 +3.616e+01 Node: ALUMINUMJJ-1.1494935 Max: +5.410e+01 Node: ALUMINUMJJ-1.3253137 4 cases 3 b 2 bar 1.5 bar normal and T only ODB: MRA_lower_3215NT.odb Abaqus/Standard 2020.HF4 Tue Mar 07 15:56:04 Eastern Standard Time/2023 4 cases 3 b 2 bar 1.5 bar normal and T only ODB: MRA_lower_3215NT.odb Abaqus/Standard 2020.HF4 Tue War 07 15 56+04 Eastern Standard Time 2023 Sten: Sten-4, thermal only 1: Step Time = 1.000 Increment Drimary Var: NT11 Step: Step-4, thermal only Increment 1: Step Time = 1.000 eformed Var: U Deformation Scale Factor: +1 Primary Var: NT11 Deformed Var: U Deformation Scale Factor: +0.000e+00

Z=0 cut through Be and Al

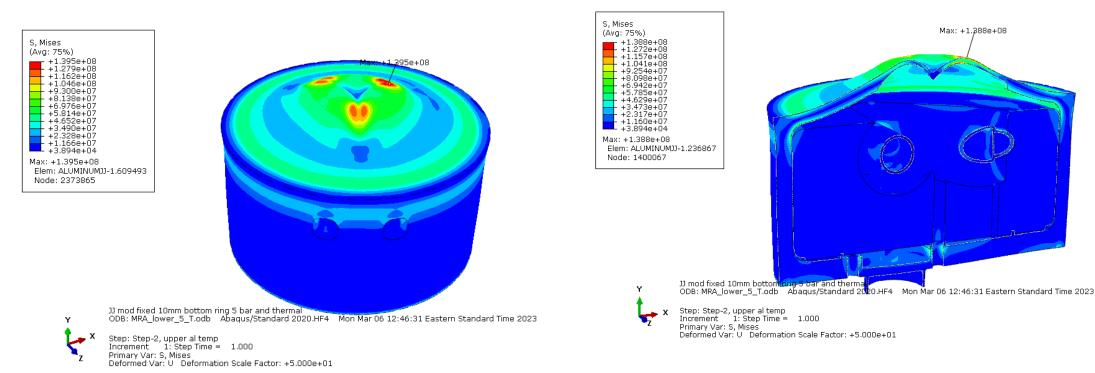
Temperature Profiles from Min-Tsung Kao – Reflector Vessel Thermal hydraulic Analysis

Min-Tsung Kao STS MRA 2022 12 07 Lower MRA Update 2



Lower Case 6 – 5 bar and thermal stress

S Mises peak 139.5 MPa



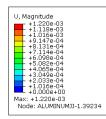
S Mises Z=0 cut

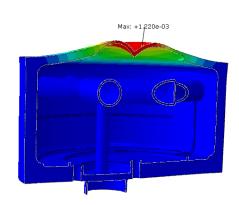
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Lower Case 6 – 5 bar and thermal stress

Displacement peak 1.2 mm

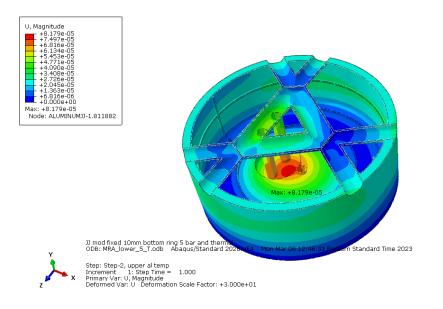




JJ mod fixed. Unmm bottom ring 5 bar and thermal
 ODB: MRA_lower_5_T.odb Abaqus/Standard 2020.HF4 Mon Mar 06 12:46:31 Eastern Standard Time 2023
 Step: Step-2, upper al temp
 Step: Step-2, upper al temp
 Torcement 1: Step Time = 1 000

Increment 1: Step Time = 1.000 Primary Var: U, Magnitude Deformed Var: U Deformation Scale Factor: +3.000e+01

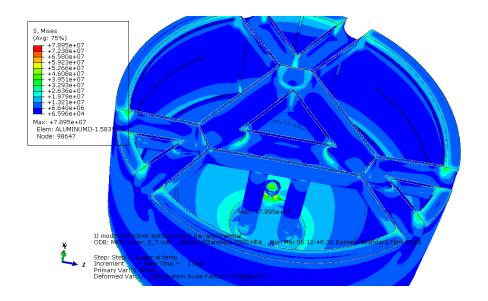
Displacement cut through neutron beam channels



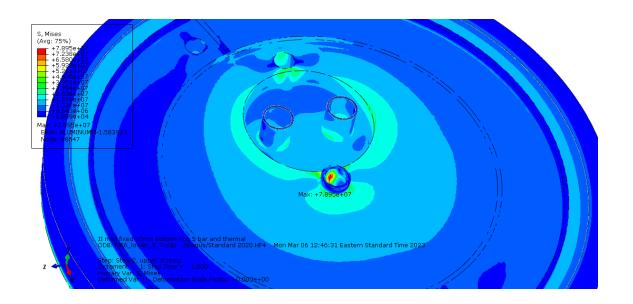


Lower Case 6-5 bar and thermal Aluminum

Y cut – S max 79 MPa near base



Peak stress near base





Summary Aluminum Peak stresses for Upper and Lower assemblies

Case	Load Case	Re	flector	Vac	Core	Therma	Allowable	Upper	Lower	Allowable	Upper	Lower	Upper	Lower
#		Ρ	(bar)		Vessel P (bar)		Aluminum Von Mises stress (MPa)	Peak	Vessel Peak Von Mises stress (MPa)		vessel Displacem ent (mm)	vessel displacem ent (mm)	vessel minimum dist. Limit to weld (mm)	
1	Reflector MAWP		5	0	0	off	127/82.7	126.3	125.4	1	1.06	1.13	8.7	15
2	Vac MAWP		3	2	0	off	127/82.7	75.8	75.2	1	.24	0.68	N/A	N/A
3	CV MAWP		3	0	1.5	off	127/82.7	73.5	37.6	1	.38	0.34	N/A	N/A
4	Thermal		0	0	0	on	254/165*	89	68.7	1	.09	.09	N/A	N/A
5	normal ops		3	0	0	on	254/165*	92.6	91.4	1	.70	0.77	N/A	N/A
6	Reflector +thermal		5	0	0	on	254/165*	134	139.5	1	-1.12	1.22	N/A	N/A

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* 93° C Primary+Bending+Secondary (3Sm) : non-weld/weld

Beryllium Peak Von Mises Stress Summary

Case #	Load Case	Upper Vessel Be peak stress (MPa)	Lower Vessel Be peak stress (MPa)
1	Reflector MAWP	30.3	27.5
2	Vac MAWP	8	15.5
3	CV MAWP	16.9	9.4
4	Thermal	66.7	50.2
5	normal ops	66.7	50.2
6	Reflector +thermal	66.7	50.2

In all cases the peak stress was well below the tensile yield strength of 345 to 517 MPa



Reflector Vessel Summary

- The design meets nearly all structural requirements with a minor exception for the 1 mm deflection limit towards the target
 - The deflections are over the limit by only ~0.2 mm or less and can be compensated by slightly dishing the heads
- The addition of thermal stresses added little to the aluminum peak stresses which are dominated by the pressure loads
- The highest beryllium stresses were due to thermal effects, but were localized and only 20% or less of the tensile limit
- No stresses in Heat Affected Zones were over the ASME limits with the closest distance of the limit to a weld HAZ was approximately 6 mm
- More realistic boundary conditions will be used in final design as the surrounding design matures

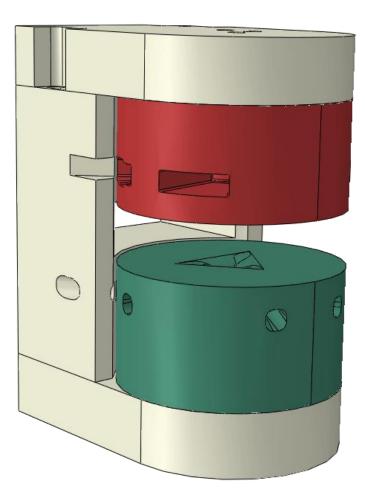


MRA Backbone Structural Analysis

- A preliminary stress analysis was done for the three lower stainless steel backbone sections without Moderator Vessels
- The principal result of concern was displacements which could affect moderator location and a limit of 0.5 mm at the center of the mounting surfaces for the reflector vessels was desired
- A full CFD analysis by Min-Tsung Kao had been done and a data file for the temperatures in the region of interest was provided
- Analysis was done for 5 bar MWAP for the water and vacuum in the core vessel. A load case was also done for a failed moderator boundary with 2 bar pressure on the internal vacuum boundary



Lower Backbone Shown with Moderator Reflector Vessels



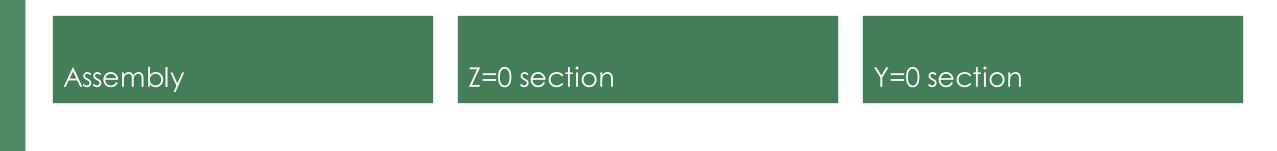
Backbone Abaqus material input data – bi-linear elasticplastic model*

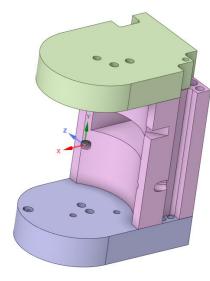
** MATERIALS
**
*Haterial, name=SS3161
*Densit
7969.,
*Elastic
1.95e+11, 0.27
*Expansion, zero=20.
1.61e-05,
*Plastic
2.5e+08, 0.
2.55e+08, 0.

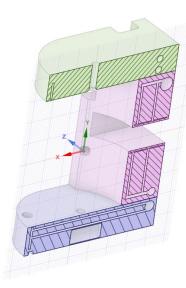
Assembly showing moderator reflector vessels alignment and material assumptions



Backbone SpaceClaim Model Section views





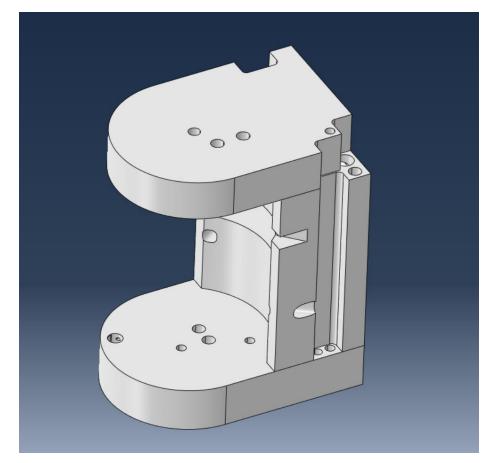




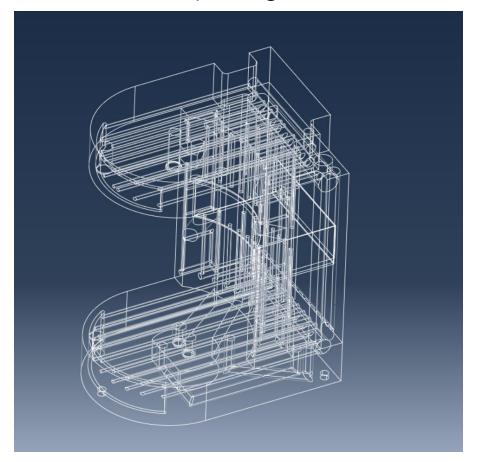


Abaqus Model with all 3 parts merged

Full model



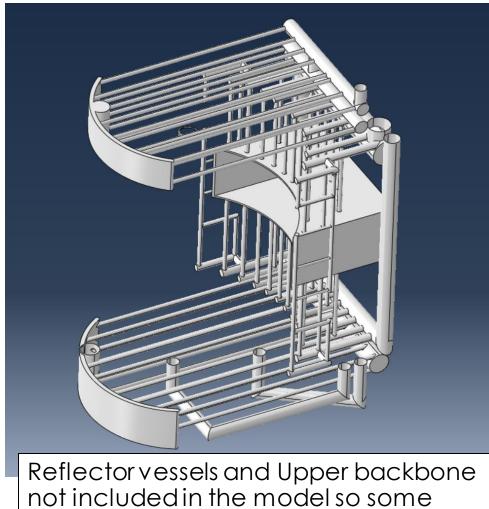
Wire Frame view showing internal passages





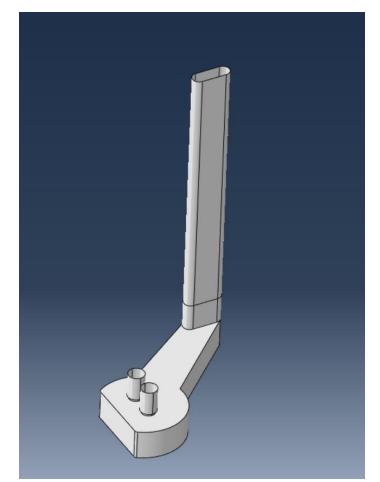
Pressure boundary surfaces

5 Bar water surfaces



pressure boundaries are not closed

Vacuum or 2 Bar Surface

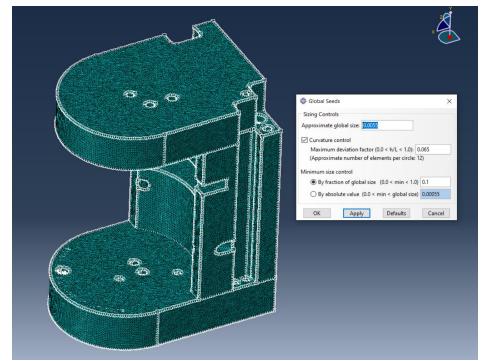




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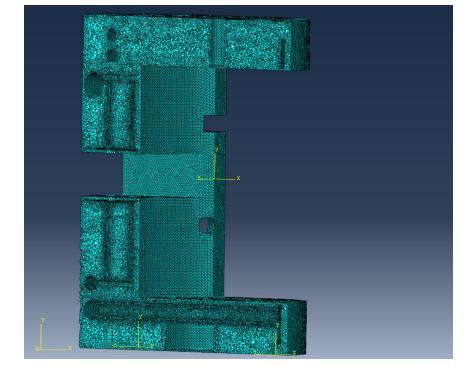
Model Mesh with C3D10 Tet elements

Nominal 5.5 mm size with 12 elements around circles



Total 3,731,555 elements

Cut near Z=0 plane



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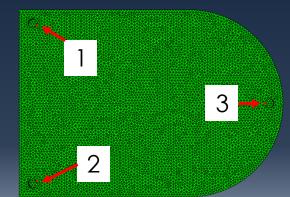
Boundary Conditions



- Boundary condition to simulate "V" groove restraint on pins in holes to force displacement in direction of central axis
- Fixed vertically at the 3 nodes

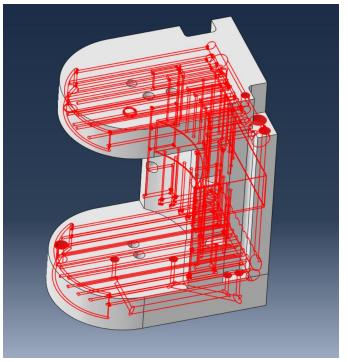
41

 Applied to one node on bottom surface of each hole

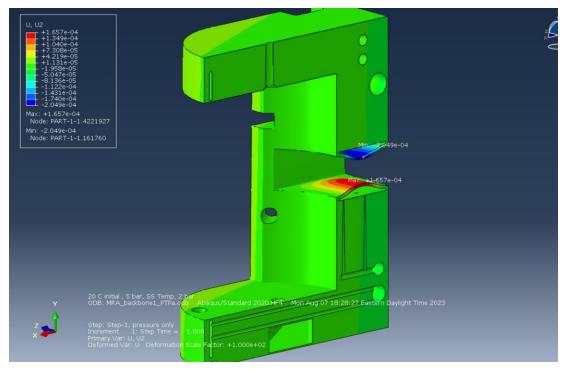


Results with 5 bar pressure and all nodes at reference temperature of 20 C

Assembly with 5 bar load applied



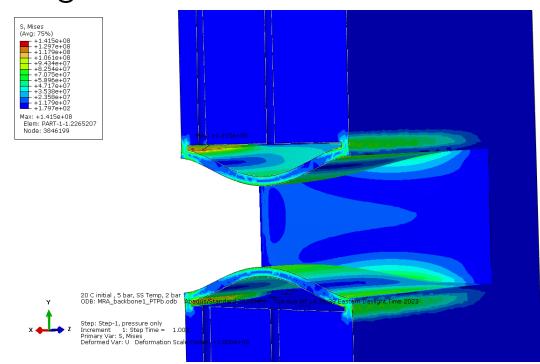
Vertical displacement - peak ~ 0.2 mm on thin walls by Proton Beam channels



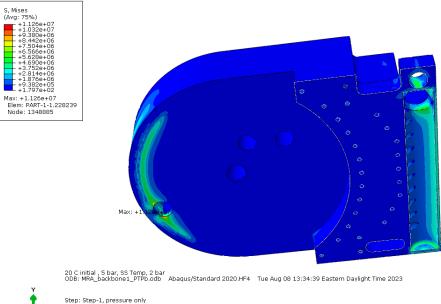


5 bar Pressure only results

S Mises 141 MPa peak by thin wall edge



Stresses in Top zone low ~ 11 MPa



Increment 1: Step Time = 1.000 Primary Var: S, Mises Deformed Var: U Deformation Scale Factor: +1.000e+02

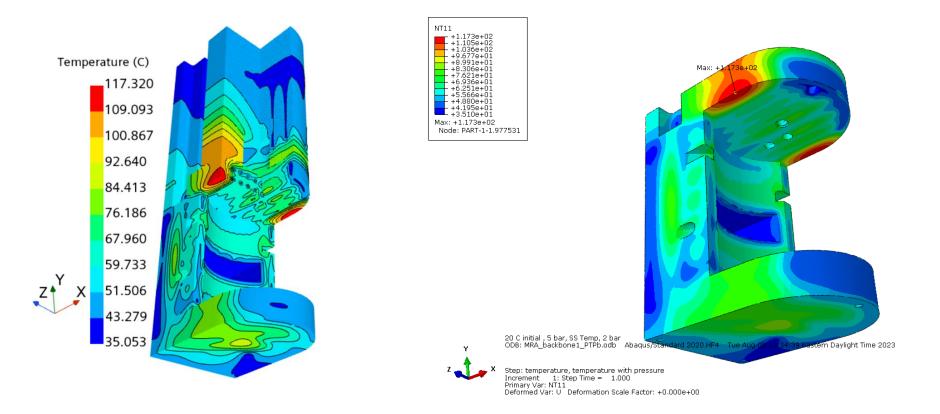
CAK RIDGE National Laboratory

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Temperature Profiles

Full Backbone Temperature profile with 0.1mm helium gap

Abaqus Mapped Temperatures – 117 C peak



M.Kao, 7/25/23 CFD Model update



Temperature and 5 bar Pressure

opening

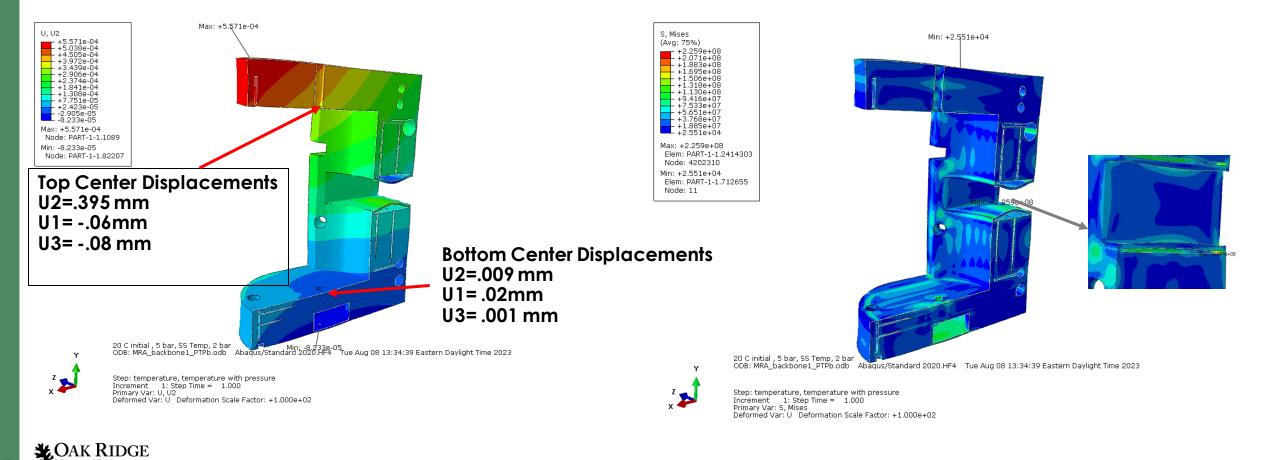
Peak Von Mises peak stress 226 MPa

in sharp corner by proton beam

Vertical Displacement < 0.5 mm on axis on upper zone

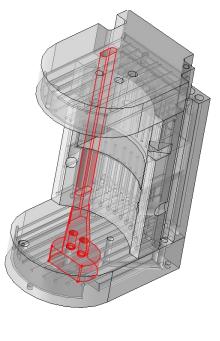
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National Laboratory

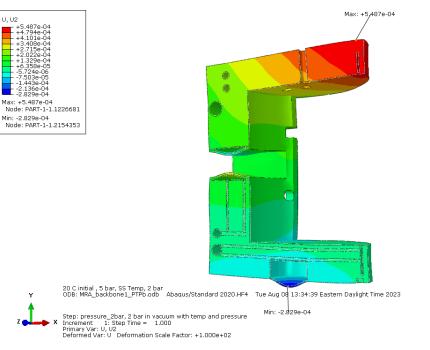


5 bar , Temperature Distribution and 2 bar pressure

Model with additional 2 bar pressure



Vertical Displacement – additional 0.28 mm on lower surface moderator zone not changed





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Backbone Results

- Top center reflector vessel mounting surface peak deflection was 0.4 mm on axis with temperature profile and 5 bar internal cooling pressure compared to a limit of 0.5 mm
- Bottom center reflector vessel mounting surface peak deflection was approximately 0.02 mm
- Peak Von Mises stress in thin wall section by beam opening was 141 MPa with 5 bar internal stress
- Peak stress of 226 MPa at localized corner near beam opening
- 2 bar pressure in vacuum region from hydrogen release did not change mounting surface deflection



Backbone Summary

- Center mounting surface deflections for normal operation due to water pressure and the temperature profile are below the 0.5 mm limit
- Final design will adjust cooling pattern to reduce rotation of top reflector
- Peak stresses for normal operation are around the thin proton beam channel wall at a sharp corner. Peak model value was 226 MPa but this was at a singularity location and no mesh refinement was attempted.
- Final design is expected to improve proton beam tube region by adding corner radii and increasing wall thickness to reduce high stresses
- Final design is also expected to include kinematic mounts and improve other regions to reduce high stresses
- The 2 bar pressure excursion stress on the lower surface was approximately 200 MPa and the mounting location deflections were not affected by this pressure release

